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# A micropolar shell model for hard-magnetic soft materials

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#### Abstract

Hard-magnetic soft materials (HMSMs) are particulate composites that particles with high coercivity are dispersed in a soft matrix. Since applying the magnetic loading induces a body couple in HMSMs, the resulting Cauchy stress is predicted to be asymmetric. Therefore, the micropolar continuum theory can be employed to capture the deformation of these materials. On the other hand, the geometries and structures made of HMSMs often possess small thickness compared to the overall dimensions of the body. Accordingly, in the present contribution, a 10-parameter micropolar shell formulation to model the finite elastic deformation of thin hard-magnetic soft structures under magnetic stimuli is developed. The proposed shell formulation allows for using three-dimensional constitutive laws without any need for modification to apply the plane stress assumption in thin structures. A nonlinear finite element formulation is also presented for the numerical solution of the governing equations. To alleviate the locking phenomenon, the enhanced assumed strain method is employed. Several examples are presented that demonstrate the performance and effectiveness of the proposed formulation.

#### KEYWORDS

magneto-elasticity, micropolar, 10-parameter shell model, HMSM, FEM

### 1 | INTRODUCTION

Magneto-active soft materials consist of magnetic particles dispersed into a soft elastomeric matrix and undergo large deformations under magnetic loading. This class of materials has been used in vibration absorbers, sensors, actuators, soft robots, flexible electronics, and isolators (see, e.g., References 1-6 and references therein). Therefore, developing reliable theoretical models plays an essential role in the optimum and cost-effective design of the aforementioned devices and instruments.

Based on the type of the embedded particles, magneto-active soft materials are divided into two sub-classes, namely soft-magnetic soft materials (SMSMs) and hard-magnetic soft materials (HMSMs). The former contains particles with low coercivity, such as iron or iron oxides, and their magnetization vector varies under external magnetic loading. This sub-class has been the subject of a huge amount of research work in this century (e.g., References 7-13). The latter sub-class is composed of particles of high coercivity, such as  $CoFe_2O_4$  or NdFeB, so that their magnetization vector, or equivalently, their remnant magnetic flux, remains unchanged for a wide range of the applied external magnetic flux (e.g., References 14,15). One of the main characteristics of HMSMs is that external magnetic induction of relatively small

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magnitude causes rapid finite deformations in these materials (e.g., References 16,17). Moreover, the 3D printing technologies have enabled the researchers to program the ferromagnetic domains in complex structures, which leads to the desired deformations References 18-22.

Theoretical modeling of HMSMs has been the subject of a plethora of research articles in recent years (e.g., References 23-32). In particular, Zhao et al.<sup>24</sup> developed a continuum formulation with asymmetric Cauchy stress so that the external magnetic induction directly contributes to the expression of the stress tensor. Their theory has been the foundation for the analysis of hard-magnetic soft beams (HMSBs) in Yan et al.,<sup>33</sup> Wang et al.,<sup>34</sup> Rajan and Arockiarajan,<sup>35</sup> and Chen et al.<sup>36,37</sup> among others. The same formulation has been employed to model the deformation of magneto-active shells by Yan et al..<sup>38</sup> Dadgar-Rad and Hossain<sup>39</sup> enhanced the formulation of Zhao et al.<sup>24</sup> to account for viscoelastic effects and analyzed the time-dependent dissipative response of HMSBs. Some researchers have developed micromechanical and lattice models for HMSMs (e.g., References 28-31,40). From a different point of view, Dadgar-Rad and Hossain<sup>32</sup> focused on the well-known phenomenon that due to the presence of the remnant magnetic induction in HMSMs, applying an external magnetic loading produces a body couple that plays the role of the main driving load to generate mechanical deformation in the body (e.g., Reference 41). Moreover, the Cauchy stress losses its symmetry, as had been previously pointed out by Zhao et al.<sup>24</sup> However, instead of following the methodology advocated in Reference 24, the authors developed a formulation based on micropolar continuum theory to predict the deformation of 3D hard-magnetic soft bodies. Two significant differences between the results of the formulation of Zhao et al.<sup>24</sup> and those based on the micropolar-enhanced formulation have been expressed by Dadgar-Rad and Hossain.<sup>32</sup>

Eringen and his coworkers established the theoretical foundations of micropolar theory. 42-44 In this theory, a microstructure at each material point is considered. The microstructure can have arbitrary rigid rotations independent of the traditional motion field considered in the classical continuum. Formulations of the micropolar theory to model localized elastic-plastic deformations (e.g., References 45-51) and size-dependent elastic deformations (e.g., References 52-57) have been developed. Some formulations to model micropolar shells have been also proposed (e.g., References 58-60). Moreover, the theory has been used in the modeling of lattice structures, crystal plasticity, phononic crystals, chiral auxetic lattices, phase-field fracture mechanics, and vertebral trabecular bone. 61-66

The current research is essentially the continuation of the previous work of the authors, namely Dadgar-Rad and Hossain,<sup>32</sup> which had been developed for three-dimensional bodies. However, most bodies made of HMSMs are thin structures, and using three-dimensional elements is computationally expensive. Accordingly, the purpose of this research is to develop a micropolar-based shell model to predict the deformation of thin HMSMs. To do so, the 7-parameter shell formulation of Sansour (e.g., References 67,68) has been extended to a 10-parameter one that involves the micro-rotation of the microstructure. On the other hand, the enhanced assumed strain method (EAS) is a widely-used strategy to eliminate locking in shell structures, for example, References 69-72. Therefore, this method is adopted here to circumvent locking effects in the present micropolar shell formulation.

The next sections of this paper are as follows: The basic kinematic and kinetic relations of the micropolar continuum theory are summarized in Section 2. In Section 3, the main characteristics of HMSMs are presented. In Section 4, the kinematic equations describing a 10-parameter micropolar shell model are provided. Section 5 presents the variational formulation, followed by a FE formulation in Section 6. Numerical examples are studied in Section 7, and the paper concludes in Section 8.

**Notation**: In this article, Greek indices take 1 and 2. All upper-case and lower-case Latin indices take 1, 2, and 3. Upper-case indices with calligraphic font, for example,  $\mathcal{K}$  and  $\mathcal{L}$ , take the values specified in the corresponding equations. The repeated Latin and Greek indices obey Einstein's summation convention. If  $\mathbf{P}$  and  $\mathbf{Q}$  are two 2nd-order tensors, the tensorial products defined via the symbols  $\otimes$ ,  $\odot$ , and  $\boxtimes$  generate 4th-order tensors so that the corresponding components are given by  $(C)_{ijkl} = (\mathbf{P} \otimes \mathbf{Q})_{ijkl} = P_{ij}Q_{kl}$ ,  $(B)_{ijkl} = (\mathbf{P} \odot \mathbf{Q})_{ijkl} = P_{ik}Q_{jl}$ , and  $(C)_{ijkl} = (\mathbf{P} \boxtimes \mathbf{Q})_{ijkl} = P_{il}Q_{kj}$ , respectively. For numerical simulations, the notation  $\mathbb{U} = \{U_{11}, U_{22}, U_{33}, U_{12}, U_{21}, U_{13}, U_{31}, U_{23}, U_{32}\}^{\mathsf{T}}$  will be used as the  $9 \times 1$  vectorial representation of the arbitrary 2nd-order tensor  $\mathbf{U}$ .

### 2 | A BRIEF REVIEW OF THE MICROPOLAR THEORY

The purpose of this section is to introduce some concepts and relations of the micropolar theory. The interested reader may refer to the pioneering works developed in References 43,44,46 for more details and discussions.

In this section, two coincident Cartesian coordinates  $\{X_I\}$  and  $\{x_i\}$ , with  $\{\mathbb{E}_I\}$  and  $\{e_i\}$  as the corresponding basis vectors, are considered. The center of a macro-element in the reference configuration  $\mathcal{B}_0$  is denoted by  $\mathbb{X}$ . After deformation

by the deformation mapping  $\psi$ , the center of the macro-element in the current configuration  $\mathcal{B}$  at the time t is denoted by x, so that  $x = \psi(X, t)$ . The deformation gradient  $\mathbf{F}$  is given by

$$\mathbf{F} = Grad \, \boldsymbol{\psi}, \quad F_{iJ} = \frac{\partial x_i}{\partial X_J}, \quad J = \det \mathbf{F} > 0,$$
 (1)

which can be uniquely decomposed as  $\mathbf{F} = \mathbf{R}\mathbf{U} = \mathbf{V}\mathbf{R}$ . Here,  $\mathbf{R}$  is the macro-rotation tensor, and  $\mathbf{U}$  and  $\mathbf{V}$  are the symmetric positive definite right and the left stretch tensors, respectively. For later use, the variation of the deformation gradient is written as follows:

$$\delta \mathbf{F} = \delta \mathbf{Y} \mathbf{F} \quad \text{with} \quad (\delta \mathbf{Y})_{ij} = (grad \ \delta \hat{\mathbf{u}})_{ij} = \frac{\partial}{\partial x_i} \delta \hat{\mathbf{u}}_i.$$
 (2)

Moreover,  $\hat{\mathbf{u}} = \mathbf{x} - \mathbf{X}$  and  $\delta \hat{\mathbf{u}} = \delta \mathbf{x}$  are the actual and virtual displacement fields, respectively.

As a basic assumption in the micropolar theory, there exists a microstructure inside each macro-element so that it experiences rigid micro-rotations independent of the macro-motion x. Let  $\theta = \theta_i e_i$  denote the micro-rotation pseudo-vector, and  $\theta = (\theta_i \theta_i)^{1/2}$  be its magnitude. The corresponding micro-rotation tensor, denoted by  $\tilde{\mathbf{R}}$ , can be expressed via the Euler–Rodriguez formula, namely (e.g., References 46,54)

$$\tilde{\mathbf{R}}(\theta) = \frac{1}{\theta^2} \left[ \theta^2 \mathbf{I} + \theta \sin \theta \hat{\boldsymbol{\theta}} + (1 - \cos \theta) \hat{\boldsymbol{\theta}}^2 \right],\tag{3}$$

where  $\hat{\theta} = -\mathcal{E}\theta$ , or  $\hat{\theta}_{ij} = -\epsilon_{ijk}\theta_k$ , is the skew-symmetric tensor corresponding to  $\theta$ . Moreover,  $\epsilon_{ijk}$  are the components of the alternating symbol  $\mathcal{E}$ . By defining  $\delta\theta$  as the virtual micro-rotation pseudo-vector, the variation of  $\tilde{\mathbf{R}}$  may be expressed via the following relations<sup>32,54</sup>

$$\delta \tilde{\mathbf{R}} = -\mathcal{E} \Lambda \delta \theta \tilde{\mathbf{R}} \quad \text{with} \quad \Lambda = \frac{1}{\theta^3} \left[ \theta^2 \sin \theta \mathbf{I} + \theta (1 - \cos \theta) \hat{\boldsymbol{\theta}} + (\theta - \sin \theta) \theta \otimes \theta \right]. \tag{4}$$

The deformation gradient, in the micropolar theory, is decomposed as  $\mathbf{F} = \tilde{\mathbf{R}}\tilde{\mathbf{U}} = \tilde{\mathbf{V}}\tilde{\mathbf{R}}$ , from which it follows that (e.g., Reference 46):

$$\tilde{\mathbf{U}} = \tilde{\mathbf{R}}^{\mathsf{T}} \mathbf{F}, \quad \tilde{U}_{IJ} = \tilde{R}_{pI} F_{pJ}, \quad \tilde{\mathbf{V}} = \mathbf{F} \tilde{\mathbf{R}}^{\mathsf{T}}, \quad \tilde{V}_{ij} = F_{iQ} \tilde{R}_{jQ}.$$
 (5)

To take the gradient of the micro-rotation into account, the material wryness tensor  $\Gamma$  and its spacial counterpart  $\gamma$ , are defined by References 43,44,46,52

$$\Gamma = -\frac{1}{2}\mathcal{E} : (\tilde{\mathbf{R}}^{\top} Grad \, \tilde{\mathbf{R}}), \quad \Gamma_{IJ} = \frac{1}{2} \epsilon_{IPK} \tilde{R}_{iK} \tilde{R}_{iP,J}, \quad \boldsymbol{\gamma} = \tilde{\mathbf{R}} \Gamma \tilde{\mathbf{R}}^{\top}, \quad \gamma_{ij} = \tilde{R}_{iP} \Gamma_{PQ} \tilde{R}_{jQ}.$$
 (6)

The deformation measures  $\tilde{\mathbf{U}}$  and  $\Gamma$  are the main kinematic tensors to develop a formulation in material framework (see also, References 43,46). Combinations of Equations (2), (4), (5)<sub>1</sub>, and (6)<sub>1</sub>, furnishes the following relations for the virtual kinematic tensors  $\delta \tilde{\mathbf{U}}$  and  $\delta \Gamma$ :<sup>32</sup>

$$\delta \tilde{\mathbf{U}} = \tilde{\mathbf{R}}^{\mathsf{T}} (\delta \mathbf{Y} - \delta \hat{\boldsymbol{\omega}}) \mathbf{F}, \quad \delta \Gamma = \tilde{\mathbf{R}}^{\mathsf{T}} \operatorname{grad} \delta \boldsymbol{\omega} \mathbf{F}, \tag{7}$$

where  $\delta \hat{\boldsymbol{\omega}} = -\mathcal{E} \delta \boldsymbol{\omega}$  is the skew-symmetric tensor corresponding to  $\delta \boldsymbol{\omega}$ .

Next, in the current configuration  $\mathcal{B}$ , let  $d\mathcal{A}$  and n be an infinitesimal area element and its corresponding outward unit normal vector, respectively. In the micropolar theory, the traction  $\mathfrak{t}^{(n)}$  and the couple vector  $\mathbf{s}^{(n)}$  (as the moment per unit area) act on  $d\mathcal{A}$ . Let  $\sigma$  and  $\mathbf{m}$  be the asymmetric Cauchy stress and the asymmetric couple stress corresponding to  $\mathfrak{t}^{(n)}$  and  $\mathfrak{s}^{(n)}$ , respectively. Accordingly, the well-known Cauchy's stress principle is extended as follows (e.g., References 32,43):

$$\mathbf{t}^{(\mathrm{n})} = \boldsymbol{\sigma} \mathrm{n}, \quad t_i^{(\mathrm{n})} = \sigma_{ij} n_j, \quad \mathbf{s}^{(\mathrm{n})} = \mathbf{m} \mathrm{n}, \quad s_i^{(\mathrm{n})} = m_{ij} n_j. \tag{8}$$

For later use, the first Piola–Kirchoff stress P, the material stress  $\tilde{P}$ , the first Piola–Kirchoff couple stress M, and the material couple stress  $\tilde{M}$  are defined by

$$\{\mathbf{P}, \mathbf{M}\} = J\{\boldsymbol{\sigma}, \mathbf{m}\}\mathbf{F}^{-\mathsf{T}}, \quad \{\tilde{\mathbf{P}}, \tilde{\mathbf{M}}\} = \tilde{\mathbf{R}}^{\mathsf{T}}\{\mathbf{P}, \mathbf{M}\} = J\tilde{\mathbf{R}}^{\mathsf{T}}\{\boldsymbol{\sigma}, \mathbf{m}\}\mathbf{F}^{-\mathsf{T}}. \tag{9}$$

#### 3 | BASIC RELATIONS OF HMSMS

The main property of hard-magnetic soft materials is the existence of a remnant magnetic flux density, that remains almost unchanged under a wide range of the applied external magnetic flux  $\mathbb{B}^{ext}$  (e.g., References 16,17,24). Let  $\tilde{\mathbb{B}}^{rem}$  and  $\mathbb{B}^{rem}$  be the remnant magnetic flux in the reference and current configurations, respectively. The relation between  $\tilde{\mathbb{B}}^{rem}$  and  $\mathbb{B}^{rem}$  is as follows:<sup>24</sup>

$$\mathbb{B}^{\text{rem}} = J^{-1} \mathbf{F} \tilde{\mathbb{B}}^{\text{rem}}, \quad B_i^{\text{rem}} = J^{-1} F_{iJ} \tilde{B}_J^{\text{rem}}. \tag{10}$$

The action of  $\mathbb{B}^{\text{ext}}$  on  $\mathbb{B}^{\text{rem}}$  leads to a body couple (moment per unit volume) in HMSMs. The relations for the body couple per unit current volume  $\mathbb{D}$ , and the body couple per unit reference volume  $\mathbb{D}^*$  may be written as (e.g., References 24,41)

$$\mathbb{p}^* = J\mathbb{p} = \frac{J}{\mu_0} \mathbb{B}^{\text{rem}} \times \mathbb{B}^{\text{ext}} = \frac{1}{\mu_0} \left( \mathbf{F} \tilde{\mathbb{B}}^{\text{rem}} \right) \times \mathbb{B}^{\text{ext}}, \tag{11}$$

where the constant  $\mu_0 = 4\pi \times 10^{-7} \frac{N}{A^2}$  is the free space magnetic permeability. For HMSMs, the external magnetic flux density  $\mathbb{B}^{\text{ext}}$  is often assumed to remain constant in space (e.g.,References 24,34-37). Using this point, it has been proven that the following Maxwell equations are satisfied in HMSMs (e.g., References 24,41):

$$Div\mathbb{B} = B_{II} = 0, \quad Curl\mathbb{H} = \epsilon_{IIK}H_{IK}\mathbb{E}_I = 0, \tag{12}$$

where  $\mathbb{B}$  is the referential magnetic flux density,  $\mathbb{H}$  is the referential magnetic field,  $\mathbb{E}_I$  is the basis vector of the referential coordinate system  $\{X_I\}$  defined in the previous section, and "Curl" is the referential curl operator.

### 4 | KINEMATICS OF A 10-PARAMETER MICROPOLAR SHELL MODEL

The geometry of a part  $\mathcal{P}$  of a shell in the reference and current configurations is displayed in Figure 1. Let  $S_0$  be the mid-surface of the shell in the reference configuration, which deforms into the surface S in the current one. As shown in Figure 1, in addition to the two common-frame Cartesian coordinates  $\{X_I\}$  and  $\{x_i\}$ , described in the previous section, the convective coordinate system  $\{\zeta^i\}$  at each material particle q of the reference mid-surface  $S_0$  is also constructed. The coordinate lines  $\zeta^i$  deform during the motion of the shell in space so that the coordinated lines  $\zeta^1$  and  $\zeta^2$  are tangent to both  $S_0$  and S. Moreover, the coordinated line  $\zeta^3 \in \left[-\frac{1}{2}h, \frac{1}{2}h\right]$ , with h as the initial thickness of the shell, is considered to be perpendicular to  $S_0$  in the reference configuration. However, it does not remain perpendicular to S in the current configuration, in general. In the sequel, for the sake of simplicity, the coordinate  $\zeta^3$  may be replaced by  $\zeta$ .

The position of the material particle q on the mid-surface  $S_0$  may be described by the vector  $\overline{\mathbb{X}}(\zeta^1, \zeta^2)$ . Let  $\{\mathbb{A}_\alpha, \mathbb{A}^\alpha, A_{\alpha\beta}, A^{\alpha\beta}, \mathbb{D}, \mathbf{B}\}$  be, respectively, the covariant and contravariant basis vectors, covariant and contravariant components of the metric tensor, outward unit normal vector, and the curvature tensor on the undeformed mid-surface  $S_0$ . Then the following relations from the differential geometry of surfaces hold (e.g., Reference 73):

$$\mathbb{A}_{\alpha} = \frac{\partial \overline{\mathbb{X}}}{\partial \zeta^{\alpha}}, \quad A_{\alpha\beta} = \mathbb{A}_{\alpha} \cdot \mathbb{A}_{\beta}, \quad A^{\alpha\eta} A_{\eta\beta} = \delta^{\alpha}_{\beta}, \quad \mathbb{A}^{\alpha} = A^{\alpha\beta} \mathbb{A}_{\beta}, \quad \mathbb{A}_{\alpha} \cdot \mathbb{A}^{\beta} = \delta^{\beta}_{\alpha} 
A = \det[A_{\alpha\beta}], \quad \mathbb{D} = \mathbb{A}_{3} = \mathbb{A}^{3} = \frac{\mathbb{A}_{1} \times \mathbb{A}_{2}}{|\mathbb{A}_{1} \times \mathbb{A}_{2}|} = \frac{\mathbb{A}_{1} \times \mathbb{A}_{2}}{\sqrt{A}}, \quad \mathbf{B} = -\mathbb{D}_{,\alpha} \otimes \mathbb{A}^{\alpha}$$
(13)

where  $\delta_{\alpha}^{\beta}$  is the two-dimensional Kronecker delta. For later use, the surface contravariant basis vectors may be written as  $\mathbb{A}^{\alpha} = A^{*\alpha J} \mathbb{E}_J$ , where  $A^{*\alpha J}$  are the Cartesian components of  $\mathbb{A}^{\alpha}$ . The position of the material particle p located at the

FIGURE 1 Geometry of the shell

elevation z with respect to  $S_0$  is described by

$$\mathbb{X}(\zeta^1, \zeta^2, z) = \overline{\mathbb{X}}(\zeta^1, \zeta^2) + z \mathbb{D}(\zeta^1, \zeta^2), \tag{14}$$

from which the covariant basis vectors  $\mathbb{G}_i$  are obtained to be

$$\mathbb{G}_{\alpha} = \frac{\partial \mathbb{X}}{\partial \zeta^{\alpha}} = \mathbb{A}_{\alpha} + z \mathbb{D}_{,\alpha}, \quad \mathbb{G}_{3} = \frac{\partial \mathbb{X}}{\partial z} = \mathbb{D}. \tag{15}$$

Motivated by Equations (13)<sub>8</sub> and (15), the symmetric shifter tensor  $\mathbf{Q} = \mathbf{Q}^{\top} = \mathbb{G}_i \otimes \mathbb{A}^i = \mathbf{I} - z\mathbf{B}$ , with  $\mathbf{I}$  as the identity tensor, is defined. The shifter tensor can be used to map the covariant and contravariant basis vectors from  $z \neq 0$  to the mid-surface with z = 0, and vice versa. More precisely, the following relations hold:

$$\mathbb{G}_i = \mathbf{Q} \mathbb{A}_i, \quad \mathbb{A}_i = \mathbf{Q}^{-1} \mathbb{G}_i, \quad \mathbb{G}^i = \mathbf{Q}^{-1} \mathbb{A}^i, \quad \mathbb{A}^i = \mathbf{Q} \mathbb{G}^i, \tag{16}$$

where  $\mathbb{G}^i$  are the contravariant basis vectors at  $\mathbb{X}$ , and use has been made of the symmetry property of  $\mathbf{Q}$ . For later use, the three-dimensional material gradient operator  $\operatorname{Grad}_{\zeta}$ , with respect to the convective coordinates  $\{\zeta^i\}$  in the reference configuration, and the material surface gradient operator  $\operatorname{Grad}_{S_0}$  with respect to  $\{\zeta^\alpha\}$  are defined as follows:

$$\operatorname{Grad}_{\zeta}\{\bullet\} = \frac{\partial\{\bullet\}}{\partial\zeta^{i}} \otimes \mathbb{G}^{i}, \quad \operatorname{Grad}_{S_{0}}\{\bullet\} = \frac{\partial\{\bullet\}}{\partial\zeta^{\alpha}} \otimes \mathbb{A}^{\alpha}.$$
 (17)

By assuming that a straight material fiber perpendicular to  $S_0$  remains straight during deformation, the following macro deformation field is considered (e.g., References 67,68):

$$\mathbf{x} = \boldsymbol{\psi}(\zeta^1, \zeta^2, z, t) = \overline{\mathbf{x}}(\zeta^1, \zeta^2, t) + z[1 + z\boldsymbol{\phi}(\zeta^1, \zeta^2, t)] d(\zeta^1, \zeta^2, t), \tag{18}$$

where  $\overline{x}$  is the image of  $\overline{X}$  on S, and d is a director vector along the deformed z-axis. Moreover, the scalar field  $\phi$  describes through the thickness stretching of the shell. Similar to the quantities defined on  $S_0$  in Equation (13), let  $\{a_{\alpha}, a^{\alpha}, a_{\alpha\beta}, a^{\alpha\beta}, n, \mathbf{b}\}$  be the surface quantities defined on S. It then follows that

$$\mathbf{a}_{\alpha} = \frac{\partial \overline{\mathbf{x}}}{\partial \zeta^{\alpha}}, \quad a_{\alpha\beta} = \mathbf{a}_{\alpha} \cdot \mathbf{a}_{\beta}, \quad a^{\alpha\eta} a_{\eta\beta} = \delta^{\alpha}_{\beta}, \quad \mathbf{a}^{\alpha} = a^{\alpha\beta} \mathbf{a}_{\beta}, \quad \mathbf{a}_{\alpha} \cdot \mathbf{a}^{\beta} = \delta^{\beta}_{\alpha}$$

$$a = \det[a_{\alpha\beta}], \quad \mathbf{n} = \frac{\mathbf{a}_{1} \times \mathbf{a}_{2}}{|\mathbf{a}_{1} \times \mathbf{a}_{2}|} = \frac{\mathbf{a}_{1} \times \mathbf{a}_{2}}{\sqrt{a}}, \quad \mathbf{b} = -\mathbf{n}_{,\alpha} \otimes \mathbf{a}^{\alpha}$$

$$(19)$$

Moreover, based on Equation (18), the covariant basis vectors  $g_i$  at x are as follows:

$$g_{\alpha} = \frac{\partial x}{\partial \mathcal{L}^{\alpha}} = a_{\alpha} + z^{2} \phi_{,\alpha} d + z(1 + z\phi) d_{,\alpha}, \quad g_{3} = \frac{\partial x}{\partial z} = (1 + 2z\phi) d. \tag{20}$$

It is observed from Equation (20)<sub>2</sub> that the director  $\mathbb{Q}$  and the basis vector  $\mathbb{Q}_3$  are in the same direction. However, the normal vector  $\mathbb{Q}_3$  are not in the same direction, in general. Next, the vectors  $\mathbb{Q} = u_i \mathbb{Q}_i$  and  $\mathbb{Q} = u_i \mathbb{Q}_i$  are defined as the mid-surface and the director displacements, respectively. This allows one to write

$$\overline{x} = \overline{X} + u, \quad d = \mathbb{D} + w.$$
 (21)

From Equations (16) and (17)<sub>1</sub>, the deformation gradient tensor **F**, described in the convective coordinate system  $\{\zeta^i\}$ , takes the form

$$\mathbf{F} = \operatorname{Grad}_{\zeta} \mathbb{X} = \frac{\partial \mathbb{X}}{\partial \zeta^{i}} \otimes \mathbb{G}^{i} = g_{i} \otimes \mathbb{G}^{i} = (g_{i} \otimes \mathbb{A}^{i}) \mathbf{Q}^{-1}.$$
 (22)

In the present shell model, using Equations (15), (16)<sub>3</sub>, (20), and (22), and neglecting the higher-order terms involving  $z^2$ , the deformation gradient is approximated as follows:

$$\mathbf{F} \approx \tilde{\mathbf{F}} \mathbf{Q}^{-1} \quad \text{with} \quad \tilde{\mathbf{F}} = \mathbf{F}^{[0]} + z \mathbf{F}^{[1]},$$
 (23)

and the tensors  $\mathbf{F}^{[0]}$  and  $\mathbf{F}^{[1]}$ , with the aid of Equations (13)<sub>8</sub> and (21), are given by

$$\mathbf{F}^{[0]} = \mathbf{a}_{i} \otimes \mathbb{A}^{i} = \mathbf{a}_{\alpha} \otimes \mathbb{A}^{\alpha} + \mathbf{d} \otimes \mathbb{D} = \mathbf{I} + Grad_{S_{0}} \mathbf{u} + \mathbf{w} \otimes \mathbb{D}$$

$$\mathbf{F}^{[1]} = \mathbf{d}_{,\alpha} \otimes \mathbb{A}^{\alpha} + 2\phi \, \mathbf{d} \otimes \mathbb{D} = Grad_{S_{0}} \mathbf{w} + 2\phi(\mathbb{D} + \mathbf{w}) \otimes \mathbb{D} - \mathbf{B}$$
(24)

To circumvent numerical difficulties in finite element solution, the in-plane deformation gradient term  $\mathbf{F}^{[0]}$  is enhanced by the second-order tensor  $\overline{\mathbf{F}}$ , to be introduced in Section 6. Accordingly, the term  $\mathbf{F}^{[0]}$  in Equation (23)<sub>2</sub> is replaced by  $\mathbf{F}^{[0]} + \overline{\mathbf{F}}$ . Moreover, following Ramezani and Naghadabadi<sup>74</sup> in the context of the micropolar Timoshenko beam model, it is assumed that the micro-rotation pseudo-vector is constant along the shell thickness, namely  $\theta = \tilde{\theta}(\zeta^1, \zeta^2)$ . Accordingly, the micro-rotation tensor  $\tilde{\mathbf{R}}(\theta)$  is independent of the z coordinate. Keeping this in mind and using Equations (5)<sub>1</sub>, (6)<sub>1</sub>, (16)<sub>3</sub>, (17)<sub>2</sub>, and (23), the micropolar deformation measures  $\tilde{\mathbf{U}}$  and  $\Gamma$ , in the present shell formulation, may be written as

$$\tilde{\mathbf{U}} = \tilde{\mathbf{R}}^{\mathsf{T}} \mathbf{F}^* \mathbf{Q}^{-1} = \tilde{\mathbf{R}}^{\mathsf{T}} (\mathbf{F}^{[0]} + \overline{\mathbf{F}} + z \mathbf{F}^{[1]}) \mathbf{Q}^{-1}, \quad \Gamma = -\frac{1}{2} \mathcal{E} : \left[ \tilde{\mathbf{R}}^{\mathsf{T}} (\operatorname{Grad}_{S_0} \tilde{\mathbf{R}}) \mathbf{Q}^{-1} \right]. \tag{25}$$

where  $\mathbf{F}^* = \tilde{\mathbf{F}} + \overline{\mathbf{F}}$  is the enhanced form of  $\tilde{\mathbf{F}}$ . The present formulation with  $\{u, w, \phi, \theta\}$  as the unknown field variables may be regarded as a 10-parameter micropolar shell model. In other words, the present formulation is the extension of the classical 7-parameter shell model, with  $\{u, w, \phi\}$  as its unknowns, introduced by Sansour (e.g., References 67,68).

### 5 | VARIATIONAL FORMULATION

Let  $\delta \mathcal{U}$  be the virtual internal energy and  $\delta \mathcal{W}$  denote the virtual work of external loads. The principle of virtual work states that  $\delta \mathcal{U} - \delta \mathcal{W} = 0$ . In what follows, the expressions for  $\delta \Psi$  and  $\delta \hat{\mathcal{W}}$ , as, respectively,  $\delta \mathcal{U}$  and  $\delta \mathcal{W}$  per unit reference volume, are derived. Moreover, for the linearization purpose to be used in the next section, the increments of  $\delta \Psi$  and  $\delta \hat{\mathcal{W}}$  are also calculated.

By neglecting thermal effects, assuming that the material is hyperelastic, and to develop a material formulation, the internal energy per unit reference volume may be written as  $\Psi = \tilde{\Psi}(\tilde{\mathbf{U}}, \mathbf{\Gamma})$ . <sup>43,46</sup> Using this point and Equation (7) furnishes

$$\delta \Psi = \frac{\partial \Psi}{\partial \tilde{\mathbf{U}}} : \delta \tilde{\mathbf{U}} + \frac{\partial \Psi}{\partial \Gamma} : \delta \Gamma = \left( \tilde{\mathbf{R}} \frac{\partial \Psi}{\partial \tilde{\mathbf{U}}} \mathbf{F}^{\mathsf{T}} \right) : (\delta \mathbf{Y} - \delta \hat{\boldsymbol{\omega}}) + \left( \tilde{\mathbf{R}} \frac{\partial \Psi}{\partial \Gamma} \mathbf{F}^{\mathsf{T}} \right) : grad \, \delta \boldsymbol{\omega}. \tag{26}$$

Moreover, the constitutive equations for the pairs  $\{\tilde{\mathbf{P}}, \tilde{\mathbf{M}}\}\$  and  $\{\mathbf{P}, \mathbf{M}\}\$  are as follows:<sup>32</sup>

$$\{\tilde{\mathbf{P}}, \tilde{\mathbf{M}}\} = \left\{ \frac{\partial \Psi}{\partial \tilde{\mathbf{U}}}, \frac{\partial \Psi}{\partial \Gamma} \right\}, \quad \{\mathbf{P}, \mathbf{M}\} = \tilde{\mathbf{R}} \left\{ \frac{\partial \Psi}{\partial \tilde{\mathbf{U}}}, \frac{\partial \Psi}{\partial \Gamma} \right\}. \tag{27}$$

It is noted that Equations (26) and (27) hold for all three-dimensional micropolar hyperelastic solids. For the present shell model, first the quantities denoted by  $\delta \mathbf{Y}^{[J]}$  ( $\mathcal{J} = 1, 2$ ) are defined by

$$\delta \mathbf{Y}^{[1]} = \delta \mathbf{F}^{[0]} + \delta \overline{\mathbf{F}} + z \delta \mathbf{F}^{[1]} - \delta \hat{\boldsymbol{\omega}} \mathbf{F}^*, \quad \delta \mathbf{Y}^{[2]} = Grad_{S_o} \delta \boldsymbol{\omega}. \tag{28}$$

Next, after replacing  $\tilde{\mathbf{F}}$  by the enhanced form  $\mathbf{F}^*$ , combination of Equations (25), (26)<sub>1</sub>, (27)<sub>1,2</sub>, and (28) leads to the following expression for  $\delta\Psi$ :

$$\delta \Psi = \mathbf{P}^{[0]} : \delta \mathbf{Y}^{[1]} + \mathbf{M}^{[0]} : \delta \mathbf{Y}^{[2]} \quad \text{with} \quad \{\mathbf{P}^{[0]}, \mathbf{M}^{[0]}\} = \{\mathbf{P}, \mathbf{M}\}\mathbf{Q}^{-1}. \tag{29}$$

For linearization purpose, the increment of  $\delta\Psi$  under the increment of the field variables  $\Delta u$ ,  $\Delta w$ ,  $\Delta \phi$ , and  $\Delta \theta$  is needed. Accordingly, from Equations (26)<sub>1</sub>, (27), and (29) it is deduced that

$$\Delta\delta\Psi = \mathbf{P}^{[0]} : \Delta\delta\mathbf{H}^{[1]} + \mathbf{M}^{[0]} : \Delta\delta\mathbf{H}^{[2]} + \delta\mathbf{Y}^{[1]} : (\mathbf{C}^{[1]} : \Delta\mathbf{Y}^{[1]} + \mathbf{C}^{[2]} : \Delta\mathbf{Y}^{[2]}) + \delta\mathbf{Y}^{[2]} : (\mathbf{C}^{[3]} : \Delta\mathbf{Y}^{[1]} + \mathbf{C}^{[4]} : \Delta\mathbf{Y}^{[2]}),$$
(30)

where  $\Delta \delta \mathbf{H}^{[1]}$  and  $\Delta \delta \mathbf{H}^{[2]}$  are as follows:

$$\Delta \delta \mathbf{H}^{[1]} = \frac{1}{2} (\Delta \hat{\boldsymbol{\omega}} \delta \hat{\boldsymbol{\omega}} + \delta \hat{\boldsymbol{\omega}} \Delta \hat{\boldsymbol{\omega}}) \mathbf{F}^* - (\delta \hat{\boldsymbol{\omega}} \Delta \mathbf{F}^* + \Delta \hat{\boldsymbol{\omega}} \delta \mathbf{F}^*) 
\Delta \delta \mathbf{H}^{[2]} = -\frac{1}{2} (\Delta \hat{\boldsymbol{\omega}} Grad_{S_0} \delta \boldsymbol{\omega} + \delta \hat{\boldsymbol{\omega}} Grad_{S_0} \Delta \boldsymbol{\omega}) \right\}.$$
(31)

Moreover, the fourth-order tensors  $C^{[I]}$  (I = 1, 2, 3, 4) have the following components:

$$C_{i_{IR}I}^{[I]} = \tilde{R}_{i_{P}}\tilde{R}_{kQ}Q_{IR}^{-1}Q_{IS}^{-1}\tilde{C}_{PROS}^{[I]} \quad \text{with} \quad Q_{IR}^{-1} = (\mathbf{Q}^{-1})_{JR}, \tag{32}$$

and  $\tilde{C}_{PIOI.}^{[I]}$  are the components of the following fourth-order tensors:

$$\tilde{\boldsymbol{C}}^{[1]} = \frac{\partial^2 \Psi}{\partial \tilde{\boldsymbol{\Pi}} \partial \tilde{\boldsymbol{\Pi}}}, \quad \tilde{\boldsymbol{C}}^{[2]} = \frac{\partial^2 \Psi}{\partial \tilde{\boldsymbol{\Pi}} \partial \boldsymbol{\Gamma}}, \quad \tilde{\boldsymbol{C}}^{[3]} = \frac{\partial^2 \Psi}{\partial \boldsymbol{\Gamma} \partial \tilde{\boldsymbol{\Pi}}}, \quad \tilde{\boldsymbol{C}}^{[4]} = \frac{\partial^2 \Psi}{\partial \boldsymbol{\Gamma} \partial \boldsymbol{\Gamma}}. \tag{33}$$

In this work, the micropolar-enhanced neo-Hookean constitutive model proposed in Reference 32 is employed, according to which the free energy density is given by

$$\Psi = \left(\eta + \frac{1}{2}\mu\right)tr(\tilde{\mathbf{U}}\tilde{\mathbf{U}}^{\mathsf{T}}) - \eta tr(\tilde{\mathbf{U}}^{2}) + \frac{1}{2}\lambda(\ln J)^{2} - \mu \ln J + \frac{1}{2}\mu l^{2}tr(\Gamma\Gamma^{\mathsf{T}}),\tag{34}$$

where  $\eta$  is a material constant, and l is the well-known material length-scale parameter. From Equations (27)<sub>1</sub> and (34), the expressions for  $\tilde{\mathbf{P}}$  and  $\tilde{\mathbf{M}}$  are then calculated to be<sup>32</sup>

$$\tilde{\mathbf{P}} = (\mu + \eta)\tilde{\mathbf{U}} - \eta\tilde{\mathbf{U}}^{\mathsf{T}} - (\mu - \lambda \ln J)\tilde{\mathbf{U}}^{\mathsf{T}}, \quad \tilde{\mathbf{M}} = \mu l^{2}\Gamma.$$
(35)

Moreover, Equations (33) and (34) lead to the following 4th-order tensors  $\tilde{\boldsymbol{C}}^{[I]}$ :

$$\tilde{\boldsymbol{C}}^{[1]} = (\mu + \eta)\mathbf{I} \odot \mathbf{I} + \lambda \tilde{\mathbf{U}}^{-\top} \otimes \tilde{\mathbf{U}}^{-\top} - \eta \mathbf{I} \boxtimes \mathbf{I} - (\lambda \ln J - \mu)\tilde{\mathbf{U}}^{-\top} \boxtimes \tilde{\mathbf{U}}^{-\top}$$

$$\tilde{\boldsymbol{C}}^{[2]} = \tilde{\boldsymbol{C}}^{[3]} = 0, \qquad \tilde{\boldsymbol{C}}^{[4]} = \mu l^2 \mathbf{I} \odot \mathbf{I}$$
(36)

Next, it is recalled that  $\mathbb{B}^{\text{ext}}$  induces the body couple density  $\mathbb{p}^*$  on an HMSM (c.f. Equation (11)<sub>2</sub>). Since  $\mathbb{p}^*$  and  $\theta$  are work-conjugate quantities, the virtual external work density  $\delta \hat{W}$  and its increment may be written as

$$\delta \hat{\mathcal{W}} = \frac{1}{\mu_0} \left[ \left( \mathbf{F} \tilde{\mathbb{B}}^{\text{rem}} \right) \times \mathbb{B}^{\text{ext}} \right] \cdot \delta \boldsymbol{\theta}$$

$$\Delta \delta \hat{\mathcal{W}} = \frac{1}{\mu_0} \left[ \left( \Delta \mathbf{F}^* \mathbf{Q}^{-1} \tilde{\mathbb{B}}^{\text{rem}} \right) \times \mathbb{B}^{\text{ext}} \right] \cdot \delta \boldsymbol{\theta}$$
(37)

As will be shown in the next section, Equation (37)<sub>2</sub> leads to the expression for the load stiffness matrix. The expressions for  $\delta\Psi$ ,  $\Delta\delta\Psi$ ,  $\delta\hat{W}$ , and  $\Delta\delta\hat{W}$  given in Equations (29), (30), and (37) are the basic relations for the FE formulation presented in the next section.

#### **6** | FE FORMULATION

A nonlinear finite element formulation for the present shell model is developed in this section. Let  $S_0^e$  be a typical element in the referential mid-surface  $S_0^e$ . To perform numerical integration, the typical element is mapped to the two-dimensional parent square element  $\Box = [-1, 1] \times [-1, 1]$  in the  $\{\xi, \eta\}$  space, with  $\xi, \eta \in [-1, 1]$ . The field variables  $\{u_i, w_i, \theta_i, \phi\}$ , over the parent element  $S_0^e$ , are interpolated as follows:

$$u_i = \mathbb{N}_u \mathbb{U}_i, \quad w_i = \mathbb{N}_w \mathbb{W}_i, \quad \theta_i = \mathbb{N}_w \mathbf{\Theta}_i, \quad \phi = \mathbb{N}_\phi \mathbf{\Phi},$$
 (38)

where  $\mathbb{N}_u = \{N_u^1, N_u^2, \dots, N_u^{n_u}\}$  is a row vector containing the shape functions that interpolate the mid-surface displacement  $u_i$  over the element. Here,  $n_u$  is the number of nodes of the element that possess the  $u_i$ -DOF. Let  $U_i^I$  be the displacement component  $u_i$  at the I'th node  $(I = 1, 2, \dots, n_u)$  of the element. Accordingly,  $\mathbb{U}_i = \{U_i^1, U_i^2, \dots, U_i^{n_u}\}^{\mathsf{T}}$  is a column vector that involves all  $U_i^I$ 's over the element. Similar definitions hold for the other quantities in Equation (38). Moreover, similar relations hold for the increment  $\{\Delta u_i, \Delta w_i, \Delta \theta_i, \Delta \phi\}$  and variation  $\{\delta u_i, \delta w_i, \delta \theta_i, \delta \phi\}$  of the field variables. The generalized displacement vector  $\mathbb{V}^e$  involving all nodal DOFs of the typical element may be written as

$$\mathbb{V}_{n^e \times 1}^e = \left\{ \mathbb{U}_1^\mathsf{T}, \mathbb{U}_2^\mathsf{T}, \mathbb{U}_3^\mathsf{T}, \mathbb{W}_1^\mathsf{T}, \mathbb{W}_2^\mathsf{T}, \mathbb{W}_3^\mathsf{T}, \mathbf{\Theta}_1^\mathsf{T}, \mathbf{\Theta}_2^\mathsf{T}, \mathbf{\Theta}_3^\mathsf{T}, \mathbf{\Phi}^\mathsf{T} \right\}^\mathsf{T}, \tag{39}$$

where  $n^e = 3(n^u + n^w + n^\theta) + n^\phi$  is the number of nodal DOFs. Based on Esq. (2), (24), (28), and (38), the following relations hold:

$$\delta F_{iJ}^{[0]} = A^{*\alpha J} \mathbb{N}_{u,\alpha} \delta \mathbb{U}_{i} + \mathbb{N}_{w} D_{J} \delta \mathbb{W}_{i} = \mathbb{D}_{iJ}^{[0]} \delta \mathbb{V}^{e}$$

$$\delta F_{iJ}^{[1]} = (A^{*\alpha J} \mathbb{N}_{w,\alpha} + 2\phi D_{J} \mathbb{N}_{w}) \delta \mathbb{W}_{i} + 2d_{i} D_{j} \mathbb{N}_{\phi} \delta \mathbf{\Phi} = \mathbb{D}_{iJ}^{[1]} \delta \mathbb{V}^{e}$$

$$(\delta \hat{\boldsymbol{\omega}} \mathbf{F}^{*})_{iJ} = \epsilon_{ijk} \Lambda_{kp} F_{jJ}^{*} \mathbb{N}_{\theta} \delta \mathbf{\Theta}_{p} = \mathbb{D}_{iJ}^{[\omega F]} \delta \mathbb{V}^{e}$$

$$\delta Y_{iJ}^{[2]} = A^{*\alpha J} (\Lambda_{ip} \mathbb{N}_{\theta})_{,\alpha} \delta \mathbf{\Theta}_{p} = \mathbb{D}_{iJ}^{[2]} \delta \mathbb{V}^{e}$$

$$(40)$$

Here, the last equality in each relation indicates that all components can be expressed in terms of the generalized virtual displacement vector  $\delta_{\mathbb{V}}^{e}$ . Next, the enhanced deformation gradient tensor  $\overline{\mathbf{F}}$  is considered. Let  $\boldsymbol{\alpha} = \{\alpha_{1}, \alpha_{2}, \ldots, \alpha_{\mathcal{P}^{*}}\}^{\mathsf{T}}$  be the column vector of enhanced parameters  $\alpha_{\mathcal{P}}$ 's ( $\mathcal{P} = 1, 2, \ldots, \mathcal{P}^{*}$ ), where  $\mathcal{P}^{*}$  is the total number of enhanced parameters. The components of  $\overline{\mathbf{F}}$  and its variation/increment depend linearly on  $\alpha_{\mathcal{P}}$ 's (see, e.g., References 69-72). Here, following the notation used in Equation (40), one may write

$$\left\{\overline{F}_{IJ}, \delta \overline{F}_{IJ}, \Delta \overline{F}_{IJ}\right\} = \overline{\mathbb{D}}_{IJ}\left\{\alpha, \delta \alpha, \Delta \alpha\right\} \quad \text{or} \quad \left\{\overline{\mathbb{F}}, \delta \overline{\mathbb{F}}, \Delta \overline{\mathbb{F}}\right\} = \overline{\mathbb{B}}\left\{\alpha, \delta \alpha, \Delta \alpha\right\},\tag{41}$$

where  $\overline{\mathbb{b}}_{iJ}$  are the  $\mathcal{P}^* \times 1$  row vectors,  $\overline{\mathbb{F}}$  is the  $9 \times 1$  vectorial representation of  $\overline{\mathbf{F}}$ , and  $\overline{\mathbb{B}}$  is a  $9 \times \mathcal{P}^*$  matrix the rows of which are  $\overline{\mathbb{b}}_{iJ}$ .

Now, let  $\mathbb{B}^{(\mathcal{N})}$  ( $\mathcal{N} = 0, 1, 2$ ) and  $\mathbb{B}^{[\omega F]}$  be the  $9 \times n^e$  matrices whose rows are  $\mathbb{b}_{iJ}^{(\mathcal{N})}$  and  $\mathbb{b}_{iJ}^{[\omega F]}$ , respectively. From Equations (28), (40), and (41) it then follows that

$$\delta \mathbb{Y}^{[1]} = \widetilde{\mathbb{B}} \delta \mathbb{V} + \overline{\mathbb{B}} \delta \alpha \quad \text{and} \quad \delta \mathbb{Y}^{[2]} = \mathbb{B}^{[2]} \delta \mathbb{V} \quad \text{with} \quad \widetilde{\mathbb{B}} = \mathbb{B}^{[0]} + \mathbb{B}^{[\omega F]} + z \mathbb{B}^{[1]}. \tag{42}$$

The differential volume element  $dV_0^e$  located at the elevation z with respect to the typical element  $S_0^e$  is given by (e.g., Reference 67)

$$d\mathcal{V}_0^e = QdS_0^e dz \quad \text{with} \quad Q = \det \mathbf{Q} \quad \text{and} \quad dS_0^e = \sqrt{A}d\zeta^1 d\zeta^2. \tag{43}$$

$$\delta \mathcal{U}^{e} = \delta \mathbf{v}^{\mathsf{T}} \mathbf{F}^{\nu}_{\text{int}} + \delta \boldsymbol{\alpha}^{\mathsf{T}} \mathbf{F}^{\alpha}_{\text{int}}, \quad \delta \mathcal{W}^{e} = \delta \mathbf{\Theta}^{\mathsf{T}}_{i} \mathbf{F}^{\theta}_{\text{ext}i} = \delta \mathbf{v}^{\mathsf{T}} \mathbf{F}^{\nu}_{\text{ext}}, \tag{44}$$

where the internal force vectors  $\mathbb{F}^{\text{int}\nu}$  and  $\mathbb{F}^{\text{int}\alpha}$  are as follows:

$$\mathbb{F}_{\text{int}}^{\nu} = \int_{\mathcal{V}_0^e} \left( \tilde{\mathbb{B}}^{\mathsf{T}} \mathbb{P}^{[0]} + \mathbb{B}^{[2]\mathsf{T}} \mathbb{M}^{[0]} \right) d\mathcal{V}_0^e, \quad \mathbb{F}_{\text{int}}^{\alpha} = \int_{\mathcal{V}_0^e} \overline{\mathbb{B}}^{\mathsf{T}} \mathbb{P}^{[0]} d\mathcal{V}_0^e. \tag{45}$$

Moreover, the external force vector  $\mathbb{F}_{\text{ext}}^{\theta}$ , work conjugate to  $\Theta_i$ , is given by

$$\mathbb{F}_{\text{ext}i}^{\theta} = \frac{1}{\mu_0} \int_{V_0^{\text{e}}} \epsilon_{imj} F_{mJ} \tilde{B}_J^{\text{rem}} B_j^{\text{ext}} \mathbb{N}_{\theta}^{\mathsf{T}} d \mathcal{V}_0^{\text{e}}.$$

$$\tag{46}$$

Next, the linearized equations resulting from Equations (30), (37)<sub>2</sub>, and (44) may be written as

$$\Delta \delta \mathcal{U}^{e} - \Delta \delta \mathcal{W}^{e} = -(\delta \mathcal{U}^{e} - \delta \mathcal{W}^{e}). \tag{47}$$

The system of algebraic equations extracted from Equation (47) may be written as

$$\begin{bmatrix} \mathbb{K}_{\text{mat}}^{\nu\nu} + \mathbb{K}_{\text{geo}}^{\nu\nu} - \mathbb{K}_{\text{load}}^{\nu\nu} & \mathbb{K}_{\text{mat}}^{\nu\alpha} + \mathbb{K}_{\text{geo}}^{\nu\alpha} - \mathbb{K}_{\text{load}}^{\nu\alpha} \\ \mathbb{K}_{\text{mat}}^{\nu\alpha\mathsf{T}} + \mathbb{K}_{\text{geo}}^{\alpha\nu} & \mathbb{K}_{\text{mat}}^{\alpha\alpha} \end{bmatrix} \begin{pmatrix} \Delta_{\mathbb{V}} \\ \Delta\alpha \end{pmatrix} = - \begin{pmatrix} \mathbb{F}_{\text{int}}^{\nu} - \mathbb{F}_{\text{ext}}^{\nu} \\ \mathbb{F}_{\text{int}}^{\alpha} \end{pmatrix}, \tag{48}$$

where the subscripts "mat", "geo", and "load", represent the material, geometric, and load part of the element stiffness matrix. In particular, the material sub-matrices  $\mathbb{K}^{\nu\nu}_{mat}$ ,  $\mathbb{K}^{\nu\alpha}_{mat}$ , and  $\mathbb{K}^{\alpha\alpha}_{mat}$  in Equation (48) are as follows:

$$\mathbb{K}_{\text{mat}}^{\nu\nu} = \int_{V_0^{\epsilon}} [\widetilde{\mathbb{B}}^{\mathsf{T}} (\mathbb{C}^{[1]} \widetilde{\mathbb{B}} + \mathbb{C}^{[3]} \mathbb{B}^{[2]}) + \mathbb{B}^{[2]\mathsf{T}} (\mathbb{C}^{[2]} \mathbb{B}^{[2]} + \mathbb{C}^{[4]} \widetilde{\mathbb{B}})] d\mathcal{V}_0^{\epsilon} 
\mathbb{K}_{\text{mat}}^{\nu\alpha} = \int_{V_0^{\epsilon}} (\widetilde{\mathbb{B}}^{\mathsf{T}} \mathbb{C}^{[1]} + \mathbb{B}^{[2]\mathsf{T}} \mathbb{C}^{[4]}) \overline{\mathbb{B}} d\mathcal{V}_0^{\epsilon}, \quad \mathbb{K}_{\text{mat}}^{\alpha\alpha} = \int_{V_0^{\epsilon}} \overline{\mathbb{B}}^{\mathsf{T}} \mathbb{C}^{[1]} \overline{\mathbb{B}} d\mathcal{V}_0^{\epsilon} \right\},$$
(49)

where  $\mathbb{C}^{[J]}$  are the matrix forms of  $C^{[J]}$ . Moreover, the load sub-matrices  $\mathbb{K}^{\nu\nu}_{\text{load}}$ ,  $\mathbb{K}^{\nu\alpha}_{\text{load}}$  are given by

$$\mathbb{K}_{\text{load}}^{\nu\nu} = \int_{V_0^e} \epsilon_{ijk} Q_{JN}^{-1} \widetilde{B}_N^{\text{rem}} B_j^{\text{ext}} \mathbb{y}_k \left( \mathbb{b}_{iJ}^{[0]} + z \mathbb{b}_{iJ}^{[1]} \right) d\mathcal{V}_0^e \\
\mathbb{K}_{\text{load}}^{\nu\alpha} = \int_{V_0^e} \epsilon_{ijk} Q_{JN}^{-1} \widetilde{B}_N^{\text{rem}} B_j^{\text{ext}} \mathbb{y}_k \overline{\mathbb{b}}_{iJ} d\mathcal{V}_0^e \right\},$$
(50)

where  $y_k = \{0_{1\times 3(n^u+n^w)}, 0_{1\times (k-1)n^\theta}, \mathbb{N}_\theta, 0_{1\times (3-k)n^\theta}, 0_{1\times n^\theta}\}^\mathsf{T}$ , with  $k\in\{1,2,3\}$ , is a column vector whose nonzero entry is  $\mathbb{N}_\theta$ . The expressions for the geometric sub-matrices, resulting from the terms  $\mathbf{P}^{[0]}: \Delta\delta\mathbf{H}^{[1]}$  and  $\mathbf{M}^{[0]}: \Delta\delta\mathbf{H}^{[2]}$  in Equation (30), are too lengthy and are not presented here. The assembled system of equations is of the form  $\mathbb{K}\Delta\mathbb{V}=-\mathbb{K}$ , where  $\mathbb{K}$ ,  $\Delta\mathbb{V}$ , and  $\mathbb{K}$  are the assembled forms of the stiffness matrix, incremental generalized displacement, and residual vector, respectively. After finding  $\Delta\mathbb{V}$ , the non-rotational quantities are update via the relations  $\mathbb{U}+\Delta\mathbb{U}\to\mathbb{U}$ ,  $\mathbb{U}+\Delta\mathbb{U}\to\mathbb{U}$ ,  $\mathbb{U}+\Delta\mathbb{U}\to\mathbb{U}$ , where  $\mathbb{U}$  is an  $\mathbb{U}+\Delta\mathbb{U}\to\mathbb{U}$ . However, the update procedure for the rotation pseudo-vector is completely different. Let  $\Delta\theta$  be the increment of the rotation pseudo-vector. The updated rotation pseudo-vector  $\theta^*_{\mathrm{updated}}$  resulting from the two subsequent rotations  $\theta$  and  $\Delta\theta$  is then calculated via the following relations:

$$\theta_{\text{updated}}^* = \frac{\theta^* + \Delta \theta^* + (\Delta \hat{\theta}^*) \theta^*}{1 - \theta^* \cdot \Delta \theta^*},\tag{51}$$

where  $\Delta \hat{\theta}^* = -\mathcal{E}\Delta \theta^*$  and  $\theta^* = \alpha \tan \frac{\theta}{2}$ . Moreover,  $\alpha = \theta/\theta$  is the unit vector along  $\theta$ . The proof of Equation (51) is lengthy and is available in, for example, Argyris.<sup>76</sup>

### 7 | NUMERICAL EXAMPLES

To examine the applicability of the developed formulation, six examples are solved in this section. The formulation has been implemented in our in-house finite element code. The 10-parameter micropolar shell element designed for the present numerical simulations is an eight-node quadrilateral. All eight nodes contain the three displacement components  $u_i$ . However, only the corner nodes contain the  $w_i$ ,  $\phi$ , and  $\theta_i$  DOFs. In other words, the DOF parameters defined after Equations (38) and (39) are  $n^u = 8$  and  $n^w = n^\theta = n^\phi = 4$ . Following Korelc and Wriggers,<sup>71</sup> the enhancing deformation gradient  $\overline{\mathbf{F}}$  is considered to be of the following form:

$$\overline{\mathbf{F}} = \mathbf{J}^{-\top} \overline{\mathbf{F}}^{\text{ref}} \mathbf{J}^{-1}, \tag{52}$$

where **J** is the Jacobi matrix between the physical and parent elements. Moreover,  $\overline{\mathbf{F}}^{\text{ref}}$  is the enhancing deformation gradient defined in the parent  $\{\xi, \eta\}$  space. In this work, the nonzero components of  $\overline{\mathbf{F}}^{\text{ref}}$  are considered as follows:

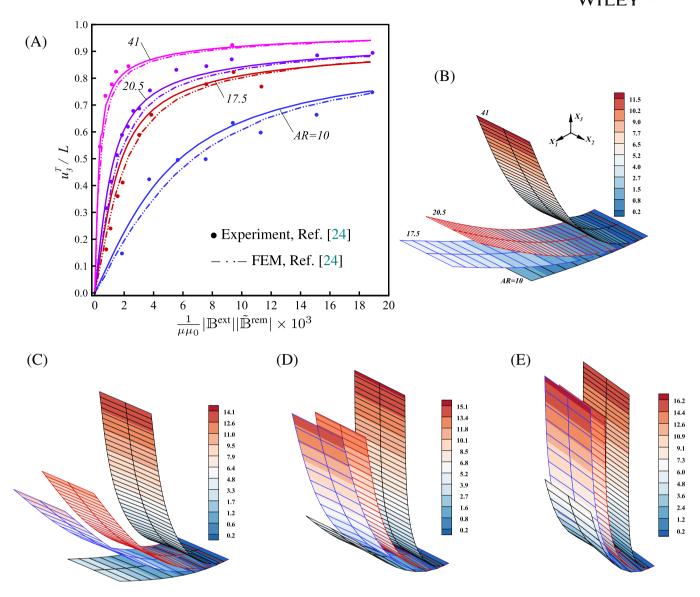
$$\overline{F}_{13}^{\text{ref}} = \alpha_1 \xi + \alpha_2 \eta, \quad \overline{F}_{23}^{\text{ref}} = \alpha_3 \xi + \alpha_4 \eta, \quad \overline{F}_{33}^{\text{ref}} = \alpha_5 \xi + \alpha_6 \eta, \tag{53}$$

which are linear functions in terms of the parent coordinates  $\xi$  and  $\eta$ . This indicates that  $\overline{\mathbf{F}}$  contains six enhanced parameters, namely  $\mathcal{P}^* = 6^*$ . To evaluate the integrals over the element surface, the  $2 \times 2$  Gauss–Legendre integration is used. Moreover, the two-point rule is employed for integration along the shell thickness.

### 7.1 | Verification Example: Bending of beam-like strips

Figure 2(A) displays the nonrmalized deflection  $u_3^{\rm T}/L$  at the tip of strips versus the nondimensional load  $\frac{10^3}{\mu\mu_0}|\mathbb{B}^{\rm rem}|$ . From the figure, it is clear that the results based on the present shell formulation are close to the numerical as well as experimental data reported in Reference 24. The deformation patterns of the strips for four values of the external magnetic flux are displayed in Figure 2(B–E). To have a comparison between the deformation of the strips for a specific value of  $|\mathbb{B}^{\rm ext}|$ , the four strips are plotted in the same figure. The importance of the aspect ratio can be observed in Figure 2(B), where the strip with AR = 41 experiences considerable large deformation even for  $|\mathbb{B}^{\rm ext}| = 2$  (mT), which is a small value for the applied magnetic flux.

It is recalled from Equation (18) that the present formulation employs the through-the-thickness stretching parameter  $\phi$ , which leads to linear shear strain as well as linear normal strain in the thickness direction. To show the effect of this parameter, two new cases are considered. In the first case, the condition  $\phi = 0$  is enforced in the formulation, while the 3D constitutive equations are still employed. In the second case, the plane stress assumption  $P_{33} = 0$  is enforced and  $\phi$  has not been considered in the formulation. Then the constitutive equation is modified to include the plane stress assumption. For the thick beam with AR = 10 and the thin one with AR = 40.5 the results are displayed in Figure 3(A,B). It is noted that for the beams with AR = 17.5 and AR = 20.5, similar results are obtained, which have not been shown in the figures. It is observed the new cases exhibit locking phenomenon in the resulting elements. The second new case is better than the first one, however, the improvement is negligible. In other words, including the through-the-thickness stretching  $\phi$  in the formulation and employing the 3D constitutive equations is an effective method for improving the performance of the present micropolar shell element.



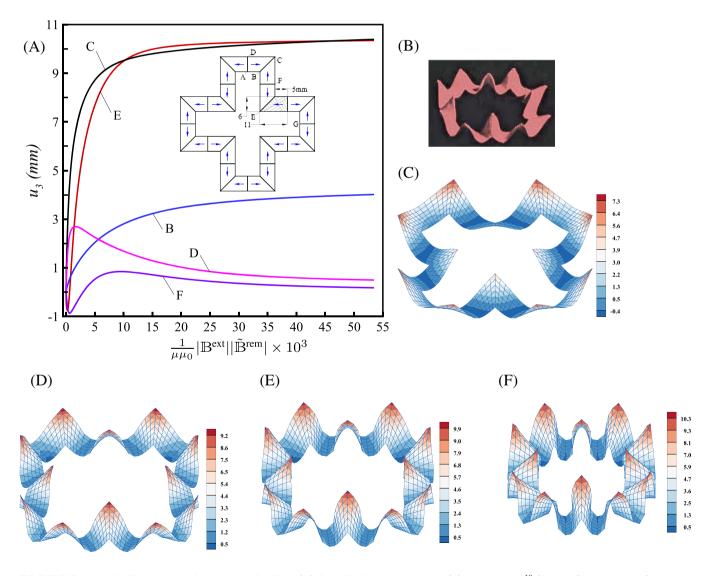
**FIGURE 2** Beam-like strips under magnetic loading, (A): the curves of  $\frac{u_3^T}{L}$  versus  $\frac{10^3}{\mu\mu_0} |\mathbb{B}^{\text{ext}}| |\tilde{\mathbb{B}}^{\text{rem}}|$ , (B–E): sequences of deformation (with the contours of  $u_3$  in mm) for  $|\mathbb{B}^{\text{ext}}| \in \{2, 5, 15, 50\}$  (mT)

### 7.2 | Deformation of a hollow cross

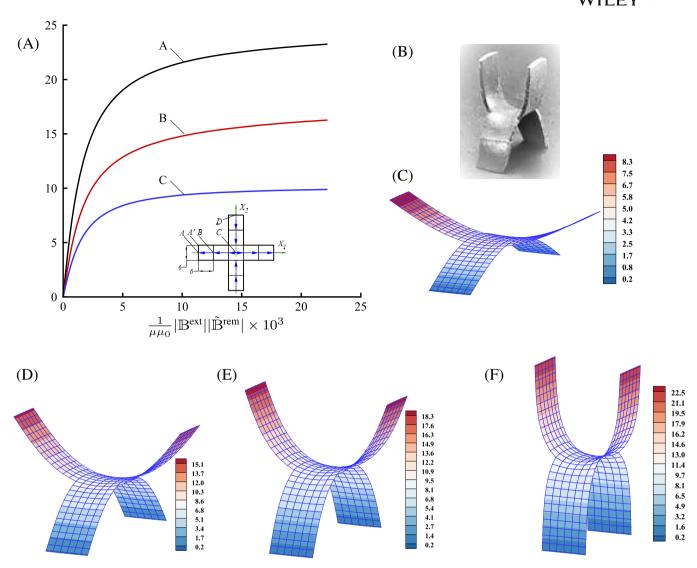
In the present example, the finite deformation of a hollow cross under magnetic loading is simulated. Following Kim et al. <sup>18</sup> and Zhao et al., <sup>24</sup> the geometry of the hollow cross is composed of 24 trapezoidal blocks (Figure 4(A)). The thickness is 0.41 mm, and all dimensions in the  $X_1X_2$  plane are displayed in the figure. The values of the mechanical properties are identical to those given in the previous example. As can be seen in Figure 4(A), the direction of  $\tilde{\mathbb{B}}^r$  is constant in each block, but it varies in different blocks. The constant value of  $|\tilde{\mathbb{B}}^r| = 102$  (mT) has been considered for the referential remnant magnetic flux. <sup>18,24</sup> The maximum external magnetic flux density  $\mathbb{B}_{\max}^{\text{ext}} = -200_{\oplus 3}$  (mT) acts on the body. The symmetry of the geometry allows us to discretize merely 1/4 of the body in the  $X_1X_2$  plane. Moreover, the displacement component  $u_3$  at the points A and G is assumed to be zero.

By performing various numerical simulations, it is found that a  $6 \times 6$  mesh of shell elements in each trapezoidal block leads to convergent results. Figure 4(A) displays the displacement component  $u_3$  against the normalized loading parameter  $\frac{10^3}{\mu\mu_0}|\mathbb{B}^{\text{ext}}||\tilde{\mathbb{B}}^{\text{rem}}|$  at some material points. At the final stage of deformation, the lateral displacement at the points E and E0 is very close to each other. More precisely, the maximum E1 displacement achieved at the point E2 is 10.39 mm. Figure 4(B) depicts the fully deformed hollow cross observed in the experiments of Kim et al. Moreover, the deformed

**FIGURE** 3 The effect of the through-the-thickness stretching parameter  $\phi$  on the load-deflection curve, (A): the thick beam with AR = 10, (B): the thin beam with AR = 41



**FIGURE 4** A hollow cross under magnetic loading, (A): load-displacement curves, (B): experiment, (C,D,E,F): sequences of deformation (with the contours of  $u_3$  in mm) for  $|\mathbb{B}^{\text{ext}}| \in \{10, 50, 100, 200\}$  (mT)



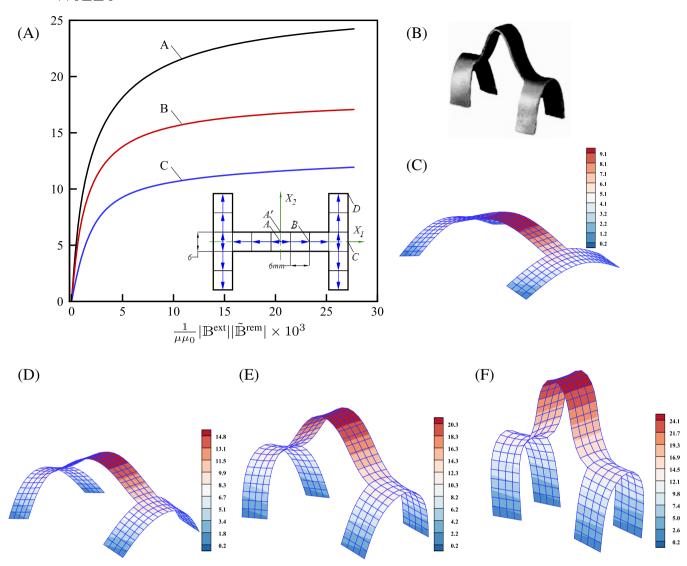
**FIGURE** 5 Deformation of a thin cross, (A): load-displacement curves, (B): experiment,  $^{20}$  (C–F): sequences of deformation (with the contours of  $u_3$  in mm) for  $|\mathbb{B}^{\text{ext}}| \in \{2, 5, 10, 40\}$  (mT)

shapes of the hollow cross under four different values of the external magnetic flux are displayed in Figure 4(C–F). By comparing Figure 4(B,E), it is deduced that the final deformed shape obtained by the present formulation is qualitatively similar to that reported in the experimental studies of Kim et al.<sup>18</sup>

### 7.3 Deformation analysis of a thin cross

The finite elastic response of a thin cross made of HMSMs is simulated in this example. The geometry of the cross involves nine welded  $6 \times 6$  (mm) square-shaped blocks (Figure 5(A)), and the thickness is 0.9 mm. The magnitude of  $\tilde{\mathbb{B}}^r$  is the constant value of 94 mT. To deform the body by magnetic loading, the maximum value of  $\mathbb{B}^{ext}$  is considered to be 40 mT, which is applied perpendicular to the plane of the cross in the  $X_3$  direction. Moreover, the mechanical properties are  $\mu = 135$  and  $\lambda = 3250$  (kPa). Due to symmetry in the  $X_1X_2$  plane, only 1/4 of the cross is used in the simulations.

From numerical experiments, it is found that a mesh containing 15 shell elements along AC and 3 elements along AA' leads to convergence in the results. The displacement component  $u_3$  at some points is plotted in Figure 5(A). As usual, the horizontal axis is considered to be the nondimensional loading  $\frac{10^3}{\mu\mu_0}|\mathbb{B}^{\text{ext}}||\tilde{\mathbb{B}}^{\text{rem}}|$ . It is noted that the  $u_3$  displacement of the point D has been considered to be zero, and the maximum lateral displacement 22.78 mm is predicted at the point A. The fully deformed shape of the cross observed experimentally in Reference 20 is displayed in Figure 5(B). Moreover, the



**FIGURE 6** Response of an H-shaped structure, (A): load-displacement curves, (B): experiment,  $^{20}$  (C–F): sequences of deformation (with the contours of  $u_3$  in mm) for  $|\mathbb{B}^{\text{ext}}| \in \{2, 5, 15, 50\}$  (mT)

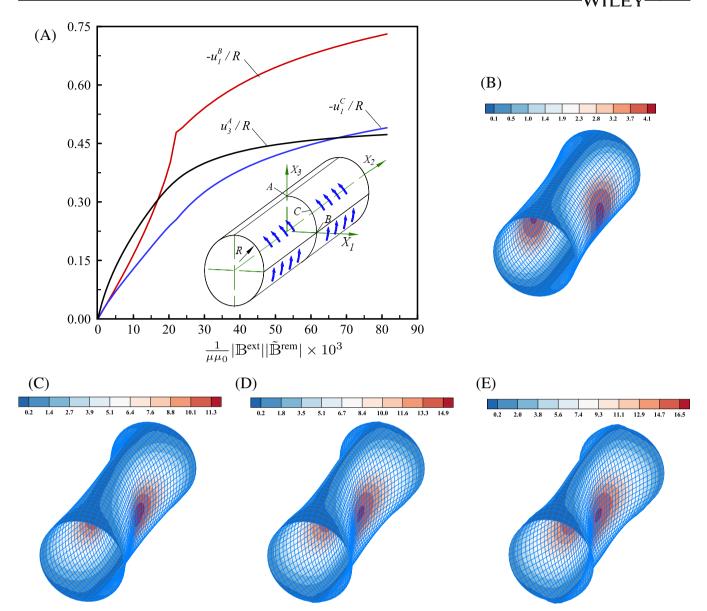
deformed shapes of the cross under four different values of the external magnetic flux are illustrated in Figure 5(C–F). Obviously, the final deformed shape in Figure 5(E), predicted by the present formulation, is qualitatively similar to that observed in the experiments of Kuang et al.<sup>20</sup> in Figure 5(B).

### 7.4 Magnetostrictive response of an H-shaped structure

In this example, the mechanical response of an H-shaped thin structure to magnetic stimuli is simulated. The geometry of the structure, consisting of fifteen blocks, is displayed in Figure 6(A). The dimensions, mechanical, and magnetic properties of the blocks are identical to those given in the example 7.3. The maximum applied magnetic loading is  $\mathbb{B}_{max}^{ext} = -50_{\oplus 3}$  (mT). By considering the symmetry properties of the geometry, merely 1/4 of the geometry is analyzed.

Numerical experiments indicate that a  $4 \times 4$  mesh of shell elements in each block yields converging results. In other words, the number of elements along AA', AC and CD is 2, 14, and 10, respectively. Figure 6(A) demonstrates the variations of the displacement component  $u_3$  at some points against the normalized loading  $\frac{10^3}{\mu\mu_0}|\mathbb{B}^{\text{ext}}||\tilde{\mathbb{B}}^{\text{rem}}|$ . By assuming zero  $u_3$  displacement at the point D, the maximum value of  $u_3 = 24.65$  mm at the point A is achieved. The fully deformed shape of the H-shaped structure from the experimental observations of Kuang et al.  $^{20}$  is illustrated in Figure 6(B). Moreover, the deformed shapes of the body under four different values of the external magnetic flux are displayed in Figure 6(C-F).



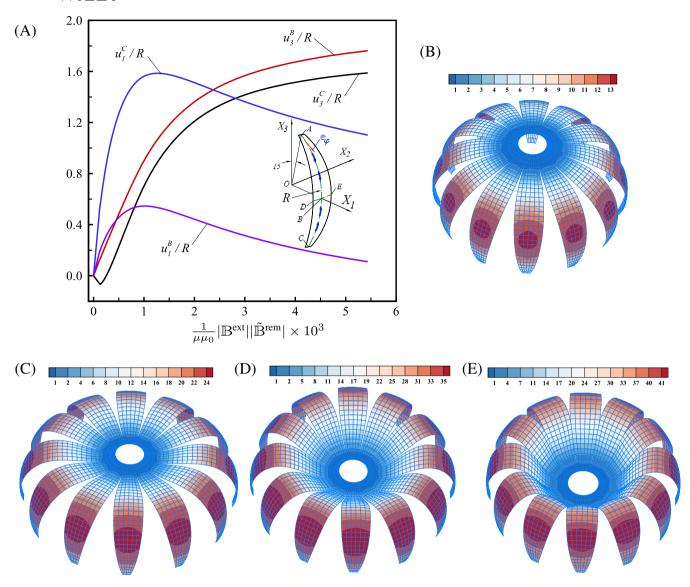


**FIGURE** 7 Deformation of a cylinder under magnetic loading, (A): load-displacement curves, (B–E): sequences of deformation (with the contours of  $|u_1|$  in mm) for  $|\mathbb{B}^{\text{ext}}| \in \{20, 50, 100, 150\}$  (mT)

A comparison of Figure 6(B,E) shows that the deformed structure obtained by the present shell formulation is qualitatively similar to that reported in the experiments of Reference 20.

### 7.5 Deformation of a cylinder (magnetic pump)

The finite elastic response of a cylindrical shell to magnetic loading is simulated in this example. As will be shown below, the deformation pattern in the cylinder is so that it may be used as a macro- or micro-fluidic magnetic pump in practical applications. In a relatively similar context, an electro-active polymer-based micro-fluidic pump can be seen in Yan et al. The present case, it is assumed that the cylinder has been made of the same blocks as described in the example 7.3. To construct the geometry, 24 blocks in the circumferential direction and 20 ones along the axis of the cylinder are used. Therefore, the mean radius and length of the cylinder are R = 22.9 and L = 120 (mm), respectively. The remnant magnetic flux  $\mathbb{B}^{\text{rem}}$  is assumed to be tangent to the cylinder surface and perpendicular to the  $X_2$  axis. Moreover, it has a positive component along the  $X_3$  axis. The magnetic flux  $\mathbb{B}^{\text{ext}}_{\text{max}} = 150\oplus_3$  (mT) acts on the cylinder. Moreover, both ends of the cylinder are considered to be clamped. Symmetry considerations allow us to simulate 1/4 of the full geometry.



**FIGURE 8** Deformation of a spherical gripper with 12 arms, (A): load-displacement curves, (B–E): sequences of deformation (with the contours of  $u_3$  in mm) for  $|\mathbb{B}^{\text{ext}}| \in \{1, 2, 4, 6.8\}$  (mT)

Numerical simulations show that a mesh of  $24 \times 20$  elements is sufficient to obtain convergent results. Variations of the displacement components  $u_1$  and  $u_3$  against  $\frac{10^3}{\mu\mu_0}|\mathbb{B}^{\rm ext}||\tilde{\mathbb{B}}^{\rm rem}|$  are plotted in Figure 7(A). The coordinates of the material points A, B and C, lying in the XZ-plane, are (0,R), (R,0), and  $\frac{1}{\sqrt{2}}(R,R)$ , respectively. The maximum (horizontal) displacement occurs at the point B and is about 16.75 mm. The deformed shapes of the cylinder under four different values of the external magnetic flux are demonstrated in Figure 6(B–E). It is observed that under the applied magnetic flux, the cylinder contracts at its middle section. This is the reason why it can be used as a magnetic pump in real applications.

## 7.6 | A magnetic gripper

The elastic response of a spherical gripper is simulated in this example. Soft grippers made of magneto-active materials have the potential as actuating components in soft robotics. For instance, Ju et al.  $^{78}$  and Carpenter et al.  $^{79}$  demonstrated additively manufactured magneto-active grippers while Kadapa and Hossain  $^{80}$  simulated the viscoelastic influences of underlying polymeric materials. In our case, the gripper is composed of 12 equal arms. In the undeformed configuration, the arms cover the surface of an incomplete sphere of radius R. It is assumed that the mechanical and magnetic properties,

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and the thickness of the HMSM are the same as those given in the example 7.3. The geometry of a single arm is shown in Figure 8(A). The arc DE lies in the  $X_1X_2$  plane, its length is 12 mm, and covers 30° of a full circle. Therefore, the mean radius of the arm is  $R = \frac{12}{\pi/6} = 22.92$  mm. The arc AC lies in the  $X_1X_3$  plane and its length is 60 mm. The angle between the radius OA and the  $X_3$ -axis is 15°, and the geometry is symmetric w.r.t. the  $X_1X_2$  plane. Moreover, the topmost arc of the arm is assumed to be clamped. As shown in the figure, let  $\mathbb{E}_{\varphi}$  be the standard meridian unit tangent vector to the sphere. It is assumed  $\mathbb{B}^{\text{rem}}$  is along  $\mathbb{E}_{\varphi}$  for  $X_3 > 0$ , and along  $-\mathbb{E}_{\varphi}$  for  $X_3 < 0$ . It is noted that applying  $\mathbb{B}^{\text{ext}}$  in  $\mathbb{E}_3$  direction opens the arms of the gripper. Here, the maximum magnetic loading  $\mathbb{B}^{\text{ext}}_{\text{max}} = 10\mathbb{E}_3$  (mT) acts on the arms.

Numerical experiments indicate that a 6 × 30 mesh of shell elements in the arm provides convergence in the results. The displacement components  $u_1$  and  $u_3$  at the points B and C versus  $\frac{10^3}{\mu\mu_0}|\mathbb{B}^{\text{ext}}||\tilde{\mathbb{B}}^{\text{rem}}|$  are plotted in Figure 8(A). For a single arm under the maximum external magnetic flux of 10 mT, the maximum value of the displacement component  $u_3$  is obtained to be about 43.9 mm. The deformed shapes of the gripper under four different values of the external magnetic flux are illustrated in Figure 8(B–E). It is noted that the maximum value of the external magnetic flux to avoid intersection between the arms is 6.8 mT. In this case, the maximum  $u_3$  component of displacement is about 41.6 mm.

### 8 | SUMMARY

In this research, a 10-parameter micropolar shell model for simulating the finite elastic deformation of thin hard-magnetic soft structures was formulated. The idea of employing the micropolar theory comes from the fact that magnetic stimulation induces a body couple on these materials, which in turn leads to asymmetric Cauchy stress tensor. Since the governing equations at finite strains, including magnetic effects, cannot be solved analytically, a nonlinear finite element formulation for simulating the problems of arbitrary thin geometry, boundary conditions, and loading cases was also presented. Six different numerical examples were solved to assess the applicability of the present formulation. It was shown that the results of the proposed formulation are in good agreement with the available experimental and numerical ones. The viscoelastic and thermal effects will be taken into account in the forthcoming contributions.

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### **CONFLICT OF INTEREST**

The authors declare no competing interests.

### DATA AVAILABILITY STATEMENT

Data sharing is not applicable to this article as no new data were created or analyzed in this study.

#### **ENDNOTE**

\*It is also possible to include the nonlinear terms involving  $\{\xi^2, \eta^2, \xi\eta^2, \xi^2\eta\}$  or  $\{1 - 3\xi^2, 1 - 3\eta^2, \xi(1 - 3\eta^2), \eta(1 - 3\xi^2)\}$  in the components of  $\overline{\mathbf{F}}^{\text{ref}}$ , which increases the number of the enhanced parameters to 39. However, our numerical simulations reveal that the change in the results is negligible.

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